

1.4. Rated Life

The service life of the Cross-Roller Ring is obtained from the following equation.

$$L = \left(\frac{f_r \cdot C}{f_w \cdot P_c} \right)^{\frac{10}{3}} \times 10^6$$

where

L : Rated life

(The total number of revolutions that 90% of a group of identical Cross-Roller Ring units independently operating under the same conditions can achieve without showing flaking from rolling fatigue)

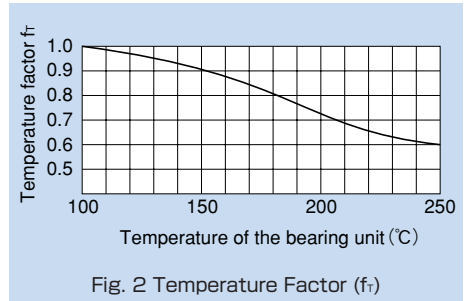
C : Basic dynamic load rating* (N)

P_c : Dynamic equivalent radial load (N)

f_r : Temperature factor (see Fig. 2)

f_w : Load factor (see table 1)

* Note: The basic dynamic load rating (C) of the Cross-Roller Ring shows the radial load with constant direction and magnitude, under which the rated life (L) is 1 million revolutions when a group of identical Cross-Roller Ring units independently operate under the same conditions. The basic dynamic load rating (C) is indicated in the "THK General Catalog - Product Specifications," provided separately.



Note: The normal service temperature is 80°C or below. If the product is to be used at a higher temperature, contact THK.

Table 1 Load Factor (f_w)

Service condition	f_w
Smooth motion without impact	1 to 1.2
Normal motion	1.2 to 1.5
Motion with severe impact	1.5 to 3

Dynamic Equivalent Radial Load P_c

The dynamic equivalent radial load of the Cross-Roller Ring is obtained from the following equation.

$$P_c = X \cdot (F_r + \frac{2M}{dp}) + Y \cdot F_a$$

where

P_c : Dynamic equivalent radial load (N)

F_r : Radial load (N)

F_a : Axial load (N)

M : Moment (N-mm)

X : Dynamic radial factor (see table 2)

Y : Dynamic axial factor (see table 2)

dp : Roller pitch circle diameter (mm)

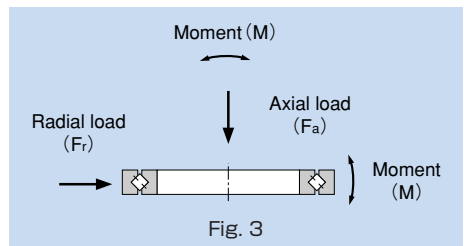


Table 2 Dynamic Radial Factor and Dynamic Axial Factor

Classification	X	Y
$\frac{F_a}{F_r + 2M/dp} \leq 1.5$	1	0.45
$\frac{F_a}{F_r + 2M/dp} > 1.5$	0.67	0.67

● If $F_r = 0$ and $M = 0$ N-mm, perform calculation while assuming that $X = 0.67$ and $Y = 0.67$.

● For service life calculation with a preload taken into account, contact THK.

[Example of calculating a service life]

Assuming that model RB25025 is used under the following conditions, calculate its rated life (L).

$$m_1 = 100 \text{ kg}$$

$$m_2 = 200 \text{ kg}$$

$$m_3 = 300 \text{ kg}$$

$$D_1 = 300 \text{ mm}$$

$$D_2 = 150 \text{ mm}$$

$$H = 200 \text{ mm}$$

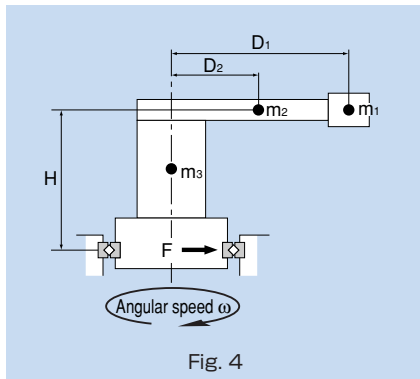
$$C = 69.3 \text{ kN}$$

$$C_0 = 150 \text{ kN}$$

$$dp = 277.5 \text{ mm}$$

$$F = 100 \text{ N}$$

$$\omega = 2 \text{ rad/s (}\omega\text{: angular speed)}$$



$$\begin{aligned} \text{Radial load : } Fr &= F + m_1 \cdot D_1 \times 10^{-3} \cdot \omega^2 + m_2 \cdot D_2 \times 10^{-3} \cdot \omega^2 \\ &= 100 + 100 \cdot 300 \times 10^{-3} \cdot 2^2 + 200 \cdot 150 \times 10^{-3} \cdot 2^2 \\ &= 340 \text{ N} \end{aligned}$$

$$\begin{aligned} \text{Axial load : } Fa &= (m_1 + m_2 + m_3) \times g \\ &= (100 + 200 + 300) \times 9.807 \\ &= 5884.2 \text{ N} \end{aligned}$$

$$\begin{aligned} \text{Moment : } M &= m_1 \cdot g \times D_1 + m_2 \cdot g \times D_2 + (m_1 \cdot D_1 \times 10^{-3} \cdot \omega^2 + m_2 \cdot D_2 \times 10^{-3} \cdot \omega^2) \times H \\ &= 100 \cdot 9.807 \times 300 + 200 \cdot 9.807 \times 150 + \\ &\quad (100 \cdot 300 \times 10^{-3} \cdot 2^2 + 200 \cdot 150 \times 10^{-3} \cdot 2^2) \times 200 \\ &= 636420 \text{ N} \cdot \text{mm} \end{aligned}$$

$$\frac{Fa}{Fr + 2M/dp} = \frac{5884.2}{340 + 2 \times 636420/277.5} = 1.19 \leq 1.5$$

$$\therefore X = 1, Y = 0.45$$

Therefore, the dynamic equivalent radial load (P_c) is obtained as follows.

$$P_c = X \left(Fr + \frac{2M}{dp} \right) + Y \cdot Fa = 1 \times \left(340 + \frac{2 \cdot 636420}{277.5} \right) + 0.45 \times 5884.2 = 7574.7 \text{ N}$$

If $f_w = 1.2$, the rated life is calculated as follows.

$$L = \left(\frac{f_r \cdot C}{f_w \cdot P_c} \right)^{\frac{10}{3}} = \left(\frac{1 \times 69.3 \times 10^3}{1.2 \times 7574.7} \right)^{\frac{10}{3}} \times 10^6 = 8.7 \times 10^8 \text{ (revolutions)}$$

Thus, the rated life (L) is 8.7×10^8 revolutions.